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**QUALITATIVE ANALYSIS
OF A
DAMPED LINEAR DYNAMIC SYSTEM**

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A THESIS

**Submitted to
Michigan State University
in partial fulfillment of the requirements
for the degree of**

MASTER OF SCIENCE

Department of Mechanical Engineering

1985

3846829

ABSTRACT

QUALITATIVE ANALYSIS
OF A
DAMPED LINEAR DYNAMIC SYSTEM

By

Jane L. Hawkins

Frequently in system analysis a qualitative description of the system response can lead to useful insight into the solution. In the case of a damped linear dynamic system, knowledge of whether the system is overdamped, underdamped or a combination may be adequate, thus making the more expensive calculation of the modes unnecessary. This thesis describes a method and presents software (QUALADS) for determining the qualitative damping nature of a system, given a system description in terms of mass, stiffness and damping matrices.

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Chapter 1

INTRODUCTION

A primary question in the analysis of a damped linear dynamic system is whether or not the system can exhibit oscillatory behavior in free motion. This question can be answered quantitatively by calculating the eigenvalues of the differential equations of motion. The process is generally complex and burdensome for large systems. The question can also be answered qualitatively by determining the system damping type. Normally a qualitative analysis requires less time and effort than a quantitative analysis, making it a more attractive method for answering the question as stated above or for gaining preliminary information about a system's dynamics.

There are several techniques for making a qualitative analysis of a damped system. The method derived by Inman and Andry [1,2] proves to be one of the more computationally efficient and comprehensive techniques. It defines a matrix, termed the critical damping matrix, which, when compared with the system's damping matrix, generates the system's damping type. This thesis describes the software QUALADS (QUALitative Analysis of Damped Systems), which uses the technique of Inman and Andry to determine the damping type of

a damped linear system described by a set of second order differential equations.

Chapter 2 describes the use of QUALADS and gives some simple examples. Chapter 3 presents an example run and users guide for QUALADS. Chapter 4 is a general discussion of quantitative versus qualitative analyses with a more detailed description of the Inman and Andry algorithm. Chapter 5 explains some of the methods of computation used by QUALADS and computation time for various segments. The final chapter provides a summary and suggestions for additional applications and extensions. The appendix contains the source code for QUALADS.

Chapter 2

QUALADS OVERVIEW

The QUALADS program performs a qualitative analysis of a damped linear dynamic system. The system must be asymptotically stable and defined by a set of second order differential equations. The inputs for QUALADS are the three coefficient matrices of the differential equations, commonly termed the mass, damping and stiffness matrices. The mass and stiffness matrices must be positive definite and symmetric and the damping matrix must be positive semi-definite and symmetric. The mass and stiffness matrices are used to compute a matrix called the critical damping matrix. By examining the matrix formed by the difference of the damping and critical damping matrices, QUALADS determines whether the system is overdamped, critically damped, underdamped or mixed damped. These damping types are the qualitative description which is the output from QUALADS. In the process of determining the damping type, QUALADS computes the undamped eigenvalues and eigenvectors as well as the critical damping matrix. If the user desires quantitative results then the damped eigenvalues may be computed as an additional option.

Consider the following 2 degree of freedom spring-mass-damper system.

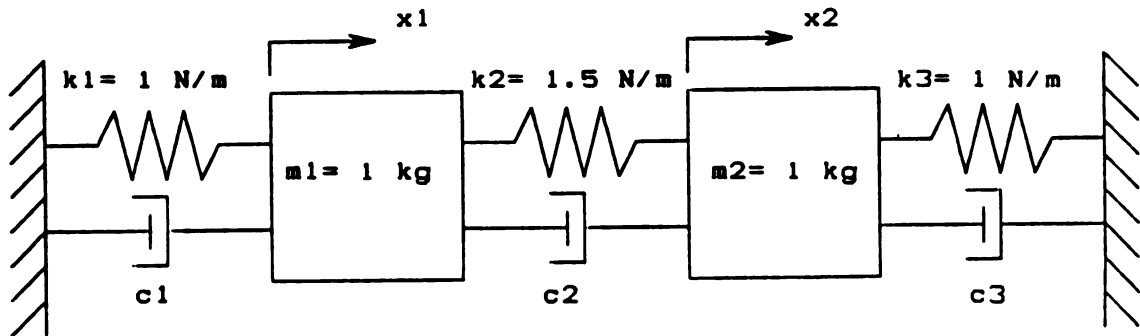


Figure 1. Two Degree of Freedom System

The mass, damping and stiffness matrices for this system are:

$$[M] = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \quad [C] = \begin{bmatrix} c1+c2 & -c2 \\ -c2 & c2+c3 \end{bmatrix} \quad [K] = \begin{bmatrix} 2.5 & -1.5 \\ -1.5 & 2.5 \end{bmatrix}$$

Depending on the values of $c1$, $c2$ and $c3$ the system will exhibit a particular damping type response.

2.1 Example 1 - Underdamped system

If $c1=c2=c3=1$ kg/sec, then the damping matrix would be:

$$[C] = \begin{bmatrix} 2 & -1 \\ -1 & 2 \end{bmatrix}$$

After inputting the mass, stiffness and damping matrices, QUALADS would respond with

```
*****
The system is underdamped
*****
```

As a check the damped eigenvalues can be computed and they are as follows.

$$\lambda_{1,2} = -0.5 \pm 0.866i$$

$$\lambda_{3,4} = -1.5 \pm 1.323i$$

Since the eigenvalues are complex, both modes will be underdamped and thus the system will be underdamped, as predicted by QUALADS.

2.2 Example 2 - Critically damped system

If $c_1=c_3=2$ kg/sec and $c_2=1$ kg/sec, then

$$[C] = \begin{bmatrix} 3 & -1 \\ -1 & 3 \end{bmatrix}$$

and QUALADS outputs

```
*****
The system is critically damped
*****
```

The damped eigenvalues for this system, as shown below, also indicate that the system is critically damped.

$$\lambda_{1,2} = -1.0$$

$$\lambda_{3,4} = -2.0$$

Since this system is critically damped, the damping matrix above must be equal to the critical damping matrix. In accord, QUALADS computes the critical damping matrix to be:

$$[C] = \begin{bmatrix} 3 & -1 \\ -1 & 3 \end{bmatrix}$$

2.3 Example 3 - Mixed damped system

Letting $c_1=c_3=2.5$ kg/sec and $c_2=.5$ kg/sec will form a damping matrix equal to:

$$[C] = \begin{bmatrix} 3 & -.5 \\ -.5 & 3 \end{bmatrix}$$

Using these damping conditions, QUALADS predicts

```
*****
The system has mixed damping
*****
```

The damped eigenvalues are

$$\lambda_1 = -.5 \quad , \quad \lambda_2 = -2.0$$

$$\lambda_{3,4} = -1.75 \pm .968i$$

Since one pair of eigenvalues is real and one pair is complex conjugate, one mode will be overdamped and one underdamped, respectively. This is mixed damping.

2.4 Example 4 - Overdamped system

Given $c_1=c_3=3$ kg/sec and $c_2=1$ kg/sec, the damping matrix will be

$$[C] = \begin{bmatrix} 4 & -1 \\ -1 & 4 \end{bmatrix}$$

These damping values when inputted into QUALADS will produce:

```
*****
The system is overdamped
*****
```

The damped eigenvalues are:

$$\lambda_1 = -.382 \quad , \quad \lambda_2 = -2.618$$

$$\lambda_3 = -1.0 \quad , \quad \lambda_4 = -4.0$$

which agrees with QUALADS conclusion.

Chapter 3

QUALADS SAMPLE RUN

This chapter provides instruction on how to use QUALADS. First an illustration of a QUALADS run will be presented. More detailed explanations of the subsections denoted by *3.2.n* appear at the end of the chapter, as well as general comments on QUALADS (section 3.2.0).

3.1 - Sample Run

Consider the two degree of freedom system with its four variants that is examined in chapter 2 (Figure 1).

$$[M] = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \quad [C] = \begin{bmatrix} c1+c2 & -c2 \\ -c2 & c2+c3 \end{bmatrix} \quad [K] = \begin{bmatrix} 2.5 & -1.5 \\ -1.5 & 2.5 \end{bmatrix}$$

- 1) $c1=c2=c3= 1 \text{ kg/sec}$
- 2) $c1=c3= 2 \text{ kg/sec}$, $c2= 1 \text{ kg/sec}$
- 3) $c1=c3= 2.5 \text{ kg/sec}$, $c2= .5 \text{ kg/sec}$
- 4) $c1=c3= 3 \text{ kg/sec}$, $c2= 1 \text{ kg/sec}$

The following QUALADS run determines the damping type and the damped eigenvalues for each of the four systems shown above. The underlined items are the information entered by the user.

OK, SEG #QUALADS

Enter the size of the system.

2

3.2.1

Enter mass matrix M by rows

ROW 1

M(1, 1)= 1

3.2.2

M(1, 2)= 0

ROW 2

M(2, 1)= 0

M(2, 2)= 1

Enter stiffness matrix K by rows

ROW 1

K(1, 1)= 2.5

K(1, 2)= -1.5

ROW 2

K(2, 1)= -1.5

K(2, 2)= 2.5

Enter the damping matrix C by rows

ROW 1

C(1, 1)= 2

C(1, 2)= -1

ROW 2

C(2, 1)= -1

C(2, 2)= 2

The system is underdamped

Do you want to compute the damped eigenvalues?

Y

3.2.3

Would you like to open a file in which to place
system matrices and output values?

Y

3.2.4

Enter filename

EXOUT

The following information can be printed
to your output file.

- 1)Mass Matrix
- 2)Stiffness Matrix
- 3)Damping Matrix
- 4)Critical Damping Matrix
- 5)Undamped Natural Frequencies
- 6)Modal Matrix
- 7)Damped Eigenvalues

Enter the option numbers desired in a single line
separated by commas.

1,2,4,5,6,3,7

3.2.5

Do you wish to run again with a different C matrix?

Y

3.2.6

Enter the damping matrix C by rows

ROW 1

C(1, 1)= 3

C(1, 2)= -1

ROW 2

C(2, 1)= -1

C(2, 2)= 3

The system is critically damped

Do you want to compute the damped eigenvalues?

Y

The following information can be printed
to your output file.

- 1)Mass Matrix
- 2)Stiffness Matrix
- 3)Damping Matrix
- 4)Critical Damping Matrix
- 5)Undamped Natural Frequencies
- 6)Modal Matrix
- 7)Damped Eigenvalues

Enter the option numbers desired in a single line
separated by commas.

3,7

Do you wish to run again with a different C matrix?
Y

Enter the damping matrix C by rows

ROW 1
 C(1, 1)= 3
 C(1, 2)= -.5

ROW 2
 C(2, 1)= -.5
 C(2, 2)= 3

 The system has mixed damping

Do you want to compute the damped eigenvalues?
Y

The following information can be printed
 to your output file.

- 1)Mass Matrix
- 2)Stiffness Matrix
- 3)Damping Matrix
- 4)Critical Damping Matrix
- 5)Undamped Natural Frequencies
- 6)Modal Matrix
- 7)Damped Eigenvalues

Enter the option numbers desired in a single line
 separated by commas.

3,7

Do you wish to run again with a different C matrix?
Y

Enter the damping matrix C by rows

ROW 1
 C(1, 1)= 4
 C(1, 2)= -1

ROW 2
 C(2, 1)= -1
 C(2, 2)= 4

 The system is overdamped

Do you want to compute the damped eigenvalues?

Y

The following information can be printed
 to your output file.

- 1)Mass Matrix
- 2)Stiffness Matrix
- 3)Damping Matrix
- 4)Critical Damping Matrix
- 5)Undamped Natural Frequencies
- 6)Modal Matrix
- 7)Damped Eigenvalues

Enter the option numbers desired in a single line
 separated by commas.

3,7

Do you wish to run again with a different C matrix?

N

*** Output data file is EXOUT

OK,

In the QUALADS run above the user requested that an output file be opened and named EXOUT. EXOUT was supplied with the data requested by the user from the menu of options presented after each system solution. The file EXOUT would appear as follows.

 Underdamped system

MASS MATRIX

Column (1)	Column (2)
0.100000D+01	0.000000D+00
0.000000D+00	0.100000D+01

STIFFNESS MATRIX

Column (1)	Column (2)
0.250000D+01	-0.150000D+01
-0.150000D+01	0.250000D+01

CRITICAL DAMPING MATRIX

Column (1)	Column (2)
0.300000D+01	-0.100000D+01
-0.100000D+01	0.300000D+01

UNDAMPED NATURAL FREQUENCIES (rad/sec)

1 0.10000D+01
 2 0.20000D+01

EIGENVECTORS (MODAL MATRIX)

Column (1)	Column (2)
0.7071	-0.7071
0.7071	0.7071

DAMPING MATRIX

Column (1)	Column (2)
0.200000D+01	-0.100000D+01
-0.100000D+01	0.200000D+01

DAMPED EIGENVALUES (Unpaired)

1 -0.500000D+00+-0.866025D+00i
 2 -0.500000D+00+ 0.866025D+00i
 3 -0.150000D+01+ 0.132288D+01i
 4 -0.150000D+01+-0.132288D+01i

 Critically damped system

DAMPING MATRIX

Column (1)	Column (2)
0.300000D+01	-0.100000D+01
-0.100000D+01	0.300000D+01

DAMPED EIGENVALUES (Unpaired)

1	-0.100000D+01+	0.000000D+00i
2	-0.100000D+01+	0.000000D+00i
3	-0.200000D+01+	0.000000D+00i
4	-0.200000D+01+	0.000000D+00i

 System with mixed damping

DAMPING MATRIX

Column (1)	Column (2)
0.300000D+01	-0.500000D+00
-0.500000D+00	0.300000D+01

DAMPED EIGENVALUES (Unpaired)

1	-0.175000D+01+-0.968246D+00i
2	-0.175000D+01+ 0.968246D+00i
3	-0.500000D+00+ 0.000000D+00i
4	-0.200000D+01+ 0.000000D+00i

 Overdamped system

DAMPING MATRIX

Column (1)	Column (2)
0.400000D+01	-0.100000D+01
-0.100000D+01	0.400000D+01

DAMPED EIGENVALUES (Unpaired)

```

1 -0.381966D+00+ 0.000000D+00i
2 -0.100000D+01+ 0.000000D+00i
3 -0.261803D+01+ 0.000000D+00i
4 -0.400000D+01+ 0.000000D+00i

```

The command file for the QUALADS run shown would be as follows.

SEG #QUALADS

```

2
1
0
0
1
2.5
-1.5
-1.5
2.5
2
-1
-1
2
Y
Y
EXOUT
1,2,4,5,6,3,7
Y
3
-1
-1
3
Y
3,7
Y
3
-.5
-.5
3
Y
3,7
Y
4
-1
-1
4
Y
3,7
N

```

3.2 - Comments and Instructions for QUALADS

3.2.0 - General Information

System Restrictions - The QUALADS algorithm has the system restrictions shown below. QUALADS checks the input matrices for their required properties and informs the user if he is attempting to run an illegal system. The program will abort if an illegal M or K matrix is entered. If an illegal C matrix is attempted or the system is asymptotically unstable the user will be asked if he wishes to enter a new C matrix.

Restrictions

M matrix - symmetric, positive definite

K matrix - symmetric, positive definite

C matrix - symmetric, positive semi-definite

System - asymptotically stable

Precision - All numbers and operations are double precision.

Yes and No - All Yes and No responses can be answered by

Y or y and N or n respectively.

Error Recovery - If an entry is not the proper data type or a valid option, the program response will be "Unrecognized Character" and "Re-enter ____".

3.2.1 - System Size

System size refers to the dimension of the square mass, stiffness and damping matrices. The maximum system size is 200.

3.2.2 - Matrix Entry

QUALADS asks for the values of the mass, stiffness, and damping matrices (in that order), element by element, working by rows.

3.2.3 - Damped Eigenvalue Option

After the damping type has been determined for a system the user has the option of computing the damped eigenvalues of the system. The damped eigenvalues are displayed in the users output file in a partially ordered fashion. The complex eigenvalues are listed first in conjugate pairs by order of magnitude, then the real eigenvalues in order of magnitude.

3.2.4 - Output File

The only information printed to the screen during a QUALADS run is the damping type. If the user wishes to examine the critical damping matrix, damped eigenvalues or other information available in the menu it has to be printed to an output file. The user is asked to enter a filename for the output file, which can be any legal filename. If the filename is already in existence QUALADS will ask if the user wishes to overwrite it.

3.2.5 - Output File Menu Options

QUALADS presents the following menu of data options that can be printed to the users designated output file.

- 1)Mass Matrix
- 2)Stiffness Matrix
- 3)Damping Matrix
- 4)Critical Damping Matrix
- 5)Undamped Natural Frequencies
- 6)Modal Matrix
- 7)Damped Eigenvalues

Enter the option numbers desired in a single line separated by commas.

As stated above the options that are desired to be printed are entered in a single line separated by commas. The options can be printed in any order and however many are desired. If no options are desired enter a blank line. The

damping type is printed to the output file regardless of the users option choice. If the user has not chosen to compute the damped eigenvalues the menu will not contain line 7.

3.2.6 - Multiple Damping Matrix Option

QUALADS has the ability to run more than one set of damping conditions for a single set of mass and stiffness conditions or matrices. After each new damping matrix is entered the damping type is determined and the user again has the option of computing the damped eigenvalues for this new system and printing to the output file. It should be noted that the only values that change for a new system are the damping matrix that was supplied by the user, the damped eigenvalues, if computed, and perhaps the damping type, depending on the system.

Chapter 4

QUALITATIVE VERSUS QUANTITATIVE ANALYSIS

4.1 - One Degree of Freedom System

In order to fully understand a qualitative analysis as compared to a quantitative analysis, a review of a simple one degree of freedom system will be helpful. The diagram below shows a spring-mass-damper system described by the differential equation noted beside it.

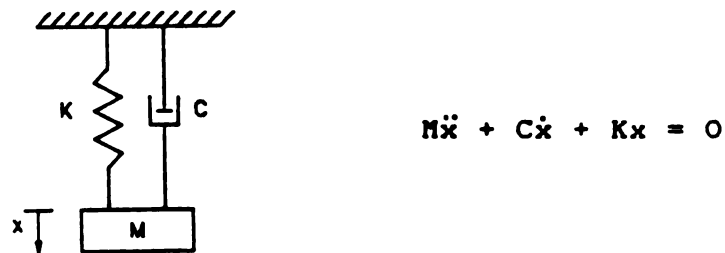


Figure 2. One Degree of Freedom System

The parameters of the system are the mass M , the spring stiffness K , and the damping constant C and are positive real constants. The system reaction to a disturbance or initial state depends on the solution or roots of the following equation called the characteristic equation.

These roots are called the damped eigenvalues of the system. They can produce three possible reactions since the two eigenvalues can be three kinds of numbers - real, real repeated or complex conjugates. Each case has a name which is called its damping 'type'. The eigenvalues and general solutions for the three cases or damping types are shown below. Note that determining the eigenvalues determines the general solution.

Damping Type	Eigenvalues	General solution
Overdamping	real $\lambda_{1,2} = -\frac{C}{2M} \pm \frac{1}{2M} \sqrt{C^2 - 4KM}$	$x(t) = A e^{\left(-\frac{C}{2M} + \frac{1}{2M} \sqrt{C^2 - 4KM}\right)t} + B e^{\left(-\frac{C}{2M} - \frac{1}{2M} \sqrt{C^2 - 4KM}\right)t}$
Critical Damping	real repeated $\lambda_{1,2} = -\frac{C}{2M}$	$x(t) = A e^{-\frac{C}{2M} t} + B t e^{-\frac{C}{2M} t}$
Underdamping	complex $\lambda_{1,2} = -\frac{C}{2M} \pm \frac{1}{2M} \sqrt{4KM - C^2} i$	$x(t) = e^{-\frac{C}{2M} t} \left(A \sin \frac{\sqrt{4KM - C^2}}{2M} t + B \cos \frac{\sqrt{4KM - C^2}}{2M} t \right)$

As can be seen by the general solutions, the underdamped system has the only oscillatory response. Knowing which of the three damping types the system exhibits tells whether the system could oscillate or not. Arriving at this classification without calculating the eigenvalues would be a

qualitative analysis of the system. The key value for determining the aforementioned is the critical damping constant C_{cr} , which is defined as follows:

$$C_{cr} = 2\sqrt{KM}$$

The relationship of the system's damping constant C to the critical damping constant C_{cr} determines the damping type as shown below.

$C > C_{cr}$	Overdamping
$C = C_{cr}$	Critical Damping
$C < C_{cr}$	Underdamping

Note that the qualitative method allows for simple and almost computation free determination of damping type if the damping constant C is changed.

For the one degree of freedom system the amount of work required in the determination of the eigenvalues for the quantitative analysis is just slightly more than the work required in comparing the critical damping to the damping constant for the qualitative analysis. The difference in workload increases when a multidegree of freedom system is examined.

4.2 - Multidegree of Freedom System

A multidegree of freedom system is described by a group of coupled second order differential equations which are expressed in matrix form below.

$$[M]\{\ddot{x}\} + [C]\{\dot{x}\} + [K]\{x\} = 0$$

The coefficient matrices are the mass matrix $[M]$, the damping matrix $[C]$ and the stiffness matrix $[K]$, all of which have real elements. If the system has N degrees of freedom, then the square matrices $[M]$, $[C]$ and $[K]$ are size $N \times N$. The unknowns are the terms of the $\{x\}$ vector where each x represents the relative motion of the system at some location.

For an N degree of freedom system there will be N pairs of eigenvalues. Each pair forms a normal mode which looks like the solution to a one degree of freedom system. Thus each mode can be defined as overdamped, critically damped or underdamped. The system's solution is a linear combination of these normal modes, where the magnitude of each mode's contribution depends on the initial conditions. If all of the normal modes are overdamped or nonoscillatory, then the system must be nonoscillatory; it is called an overdamped system. If all of the modes are critically damped, then the system is critically damped. If all of the modes are

underdamped, then the system is underdamped (oscillatory). There is of course the possibility that the modes are a mixture of the three damping types. In this case the system is said to be mixed damped.

As in the one degree of freedom system, the principal step in a quantitative analysis of a multidegree of freedom system is to calculate the system's eigenvalues. Examining the eigenvalues will tell if the system can oscillate. The most comprehensive method to solve for the eigenvalues of a size N system is to convert the system into a size $2 \times N$ system of first order state equations. The determination of the $2N$ eigenvalues requires lengthy matrix operations. Taking this into account, as well as the fact that system size has doubled, may make a quantitative analysis less attractive than a qualitative analysis in a design context.

4.3 - Qualitative Method by Inman and Andry

A qualitative analysis of a multidegree of freedom system should determine whether a system can oscillate without computing the solution, i.e. eigenvalues. A method which accomplishes this was derived by Inman and Andry [1,2]. Their algorithm defines and uses a matrix called the critical damping matrix, $[C_{cr}]$.

$$[C_{cr}] = 2[M][U][w][U^T][M]$$

The matrix $[U]$ is the undamped modal matrix normalized such that $[U]^T[M][U] = [I]$ and the matrix $[w]$ is the diagonal matrix containing the undamped natural frequencies. Both matrices come from solving the eigenvalue problem of the undamped system $[M]\{\ddot{x}\} + [K]\{x\} = 0$. The algorithm is restricted to asymptotically stable systems with symmetric, positive definite mass and stiffness matrices and a symmetric, positive semi-definite damping matrix.

The critical damping matrix serves a function similar to that of the critical damping constant in the single degree of freedom system. The damping type of a system is ascertained by determining the definiteness of the matrix equal to the difference between the damping matrix $[C]$ and the critical damping matrix $[C_{cr}]$. The relationships are as follows.

<u>$[C - C_{cr}]$</u>	<u>System damping type</u>
positive definite	overdamping
zero	critical damping
negative definite	underdamping
indefinite	mixed damping

Note again that changing the damping of the system does not affect the critical damping value. Thus to reexamine a system with a different damping matrix only requires

determining the definiteness of the new [C - Ccr]. This makes the qualitative method more attractive than the quantitative, which would require that the analysis be reworked from the beginning.

Chapter 5

QUALADS COMPUTATIONAL INFORMATION

This chapter presents a brief description of the major methods of computation used in QUALADS. It then compares the computational load for the qualitative method of determining damping type with that for the quantitative procedure of determining the damped eigenvalues.

5.1 - Methods of Computation

Eigenvalue Problem - The undamped natural frequencies, modal matrix, and damped eigenvalues are all computed using the IMSL eigenvalue package EIGZF [3]. Given the mass and stiffness matrices, EIGZF computes the undamped eigenvalues and eigenvectors, which produce the natural frequencies and normalized modal matrix. For the damped eigenvalues the system equations are converted to state space equations [4]. The "mass" and "stiffness" matrices for state space are as follows.

$$"M" = \begin{bmatrix} \text{---} & & \text{---} \\ & O & M \\ \text{---} & \text{---} & \text{---} \\ & M & C \\ \text{---} & & \text{---} \end{bmatrix}$$

$$"K" = \begin{bmatrix} \text{---} & & \text{---} \\ & -M & O \\ \text{---} & \text{---} & \text{---} \\ & O & K \\ \text{---} & & \text{---} \end{bmatrix}$$

Given the above matrices EIGZF computes the damped eigenvalues unpaired and unordered.

Definiteness - To determine the damping type QUALADS must determine if the $[C - Ccr]$ matrix is positive definite, negative definite, indefinite or zero. The procedure used involves determining the sign of the determinants of the submatrices. Let $[A] = [C - Ccr]$. For the square matrix $[A]$ that is size n , the square submatrix $[A_k]$ is a size k matrix formed by the first k rows and first k columns of the $[A]$ matrix. The following conditions determine the corresponding definite type [5].

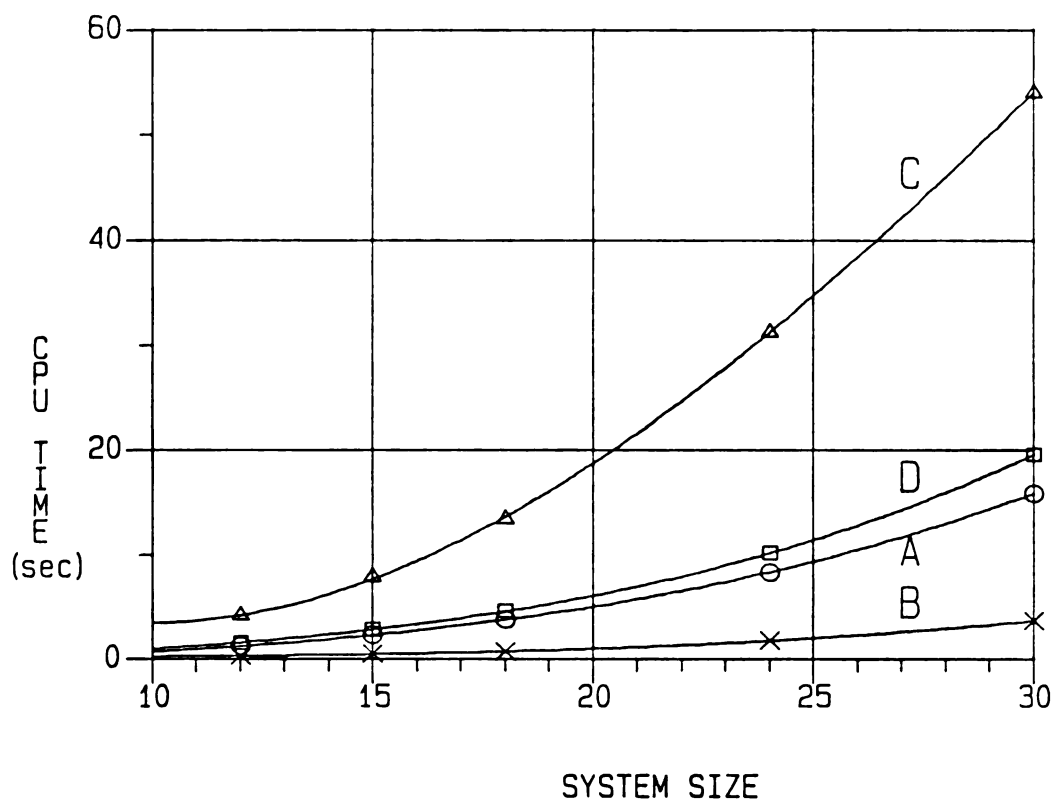
<u>Condition</u>	<u>Type</u>
$\text{Det}([A_k]) > 0$ for $k=1,2,\dots,n$	positive definite
$\text{Det}([-A_k]) > 0$ for $k=1,2,\dots,n$	negative definite

If $[A]$ is neither positive definite nor negative definite nor zero, then it is indefinite.

Asymptotic Stability - If the damping matrix $[C]$ of a system is positive semi-definite, then the system could be asymptotically unstable. QUALADS uses the criterion that a system is asymptotically unstable if any of the modal vectors of the undamped system lie in the null space of the damping matrix $[C]$ [6]. This check is achieved by examining the matrix that is the product of the damping matrix $[C]$ and the modal matrix $[U]$. If none of the columns of this product matrix are zero vectors then the system is stable.

5.2 - Computation Time

The major part of the computational load in determining the damping type occurs during the computation of the critical damping matrix $[C_{cr}]$ as opposed to determining the definiteness of the $[C - C_{cr}]$ matrix. If a set of several damping matrices is examined during one run then the computational load is considerably less after the initial system has been solved. This occurs because the critical damping matrix, which is determined by the mass and stiffness matrix, has already been computed. The computational load incurred by the quantitative analysis, which consists of computing the damped eigenvalues, is much greater than that of the qualitative analysis. Figure 3 shows the CPU (Central Processor Unit) time required to determine $[C_{cr}]$, the definiteness (damping type) given $[C]$ and $[C_{cr}]$, and the damped eigenvalues. All the calculations were for overdamped systems, which produces the maximum computational effort in the definiteness routine. Although the data suggests that CPU time is a function of the system size squared for all three computations, the controlling coefficient of the definiteness algorithm is small relative to the coefficient of the eigenvalue algorithm. Therefore, as system size increases the computational savings afforded by the qualitative method become substantial.



- A - Computation of critical damping matrix $[C_{cr}]$
- B - Computation of definiteness of $[C] - [C_{cr}]$
(damping type) given $[C_{cr}]$
- C - Computation of damped eigenvalues
- D - Computation of damping type given $[M]$, $[C]$,
and $[K]$ (i.e. A + B)

Figure 3. CPU Time Plot

These results were obtained on a Prime 750 virtual memory super mini-computer with a 6 Mb memory operating under PRIMOS Revision 19.2.

Chapter 6

SUMMARY

A qualitative analysis of damped linear dynamic systems was considered. This analysis consists of an algorithm derived by Inman and Andry that determines a matrix called the critical damping matrix, which is then used to generate the damping type of a system described by a set of linear second order differential equations. Software called QUALADS was written to implement this qualitative analysis method. In addition to finding the damping type, QUALADS can execute a quantitative analysis of a system by computing the damped eigenvalues.

6.1 - Conclusions

In the examination of a damped system the question of the system's ability to exhibit oscillatory behavior in free motion is generally an important question. The qualitative analysis described in this thesis offers an effective method for answering this question. As compared to a quantitative solution to the problem, the qualitative analysis requires less computational effort. The compute time and effort difference is even greater when several damping mechanisms for the same model are to be examined. There the qualitative

solution determined by QUALADS may yield sufficient information about a system or provide sufficient preliminary data to guide a further investigation of the damping behavior.

6.2 - Extensions

An interesting extension to the qualitative analyses examined in this thesis is the inverse problem of designing a system that will not oscillate when perturbed. For an overdamped or nonoscillatory system, the analysis requires that the matrix $[C - Ccr]$ be positive definite. This requirement would give rise to a set of nonlinear inequalities. An algorithm to solve these inequalities, along with additional user imposed constraints, could prove to be a very useful design tool.

LIST OF REFERENCES

LIST OF REFERENCES

1. Inman, D. J. and Andry, A. N., "Some Results on the Nature of Eigenvalues of Discrete Damped Linear Systems", Journal of Applied Mechanics, Dec. 1980.
2. Inman, D. J., "Nature of the Solutions of Damped Linear Dynamic Systems, Ph.D. Dissertation, Michigan State University, 1980.
3. "IMSL Library - User's Manual", IMSL Inc., Texas, 1984, Chapter E.
4. Meirovitch, L., "Analytical Methods in Vibrations", The Macmillan Co., N.Y., 1967, pg. 411,412.
5. Phillips, R., "Lecture Notes on Matrix Theory", Michigan State University, 1981.
6. Moran, T. J., "A Simple Alternative to the Routh-Hurwitz Criterion for Symmetric Systems", Journal of Applied Mechanics, Dec. 1970.

APPENDIX

The following appendix consists of the source code for QUALADS. The code is written in FORTRAN 77. The first page of code is the program MAIN which calls the main subroutines of the program. An explanation of the purpose of each subroutine is given before its call. A calling tree for the subroutines of QUALADS follows.

```

MAIN (QUALADS)
  DIMSIZ
  MATINP
  CHKM
    CHKSYM
    PDCHK
      DETER
      DIAGPR
  MATINP
  CHKK
    CHKSYM
    PDCHK
      DETER
      DIAGPR
  MATINP
  CHKC
    CHKSYM
    PDCHK
      DETER
      DIAGPR
    PSDCHK
      DETER
      DIAGPR
  EIGEN
    EIGZF (IMSL)
    SORT
    NORM
  CHKSTB
  COMP
  DEFNT
    DETER
    DIAGPR
  EIGQUE
  DMPEIG
    EIGZF (IMSL)
  OPNFIL
  OUTFIL
  RERUN

```

The subroutines shown in the calling tree appear in the code in the following order. The subroutines TRMTYP and PAGE perform screen paging for those terminals that require it and are called throughout QUALADS in several of its subroutines.

MAIN
DIMSIZ
CHKM
CHKK
CHKC
CHKSTB
EIGEN
SORT
NORM
DEFNT
EIGQUE
DMPEIG
OPNFIL
OUTFIL
RERUN
CHKSYM
PDCHK
PSDCHK
DETER
DIAGPR
TRMTYP
PAGE


```

NN=N*N
N2=2*N
NN=2*N2*N2
C
C
C*****C
C*   MATINP asks the user to input the mass, stiffness, and      *C
C*   damping matrices (M,K,C).                                     *C
C*   CHKM and CHKK check the M and K matrices, respectively, for *C
C*   symmetry and positive definiteness.                           *C
C*   CHKC checks the C matrix for symmetry and positive semi-     *C
C*   definiteness.                                                  *C
C*   CHKSTB checks to see if the system is asymptotically stable. *C
C*   (Since this check requires the modal matrix U, EIGEN is       *C
C*   is called to compute it. See EIGEN below.)                    *C
C*****C
C
  CALL MATINP(M,N,'M')
  CALL CHKM(M,N,WKL,NN,ABORT)
  IF(ABORT)GO TO 999
C
  CALL MATINP(K,N,'K')
  CALL CHKK(K,N,WKL,NN,ABORT)
  IF(ABORT)GO TO 999
C
C*****C
C
10  CONTINUE
  CALL MATINP(C,N,'C')
C
  CALL CHKC(C,N,WKL,NN,OK,PSD)
  IF(OK)THEN
    IF(PSD)THEN
      IF(.NOT.UCALC)THEN
        CALL EIGEN(M,K,W,U,N,NN2,WK1,WKL,WK1C,WK11,WK22,WK11C)
        UCALC=.TRUE.
      ENDIF
      CALL CHKSTB(C,U,WK11,N,OK)
    ENDIF
  ENDIF
C
C
C*****C
C
  IF(OK)THEN
C
C
C*****C
C*   EIGEN computes the undamped eigenvalues (natural             *C
C*   frequencies)(W) and eigen vectors (modal matrix)(U).        *C
C*   NN2 is the dimension of a workspace needed by the            *C
C*   eigenvalue package EIGZF (IMSLD).                             *C
C*****C
C
  IF(.NOT.UCALC)THEN
    CALL EIGEN(M,K,W,U,N,NN2,WK1,WKL,WK1C,WK11,WK22,WK11C)
    UCALC=.TRUE.

```

```

      ENDIF

C
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C*****
C*      COMP computes the critical damping matrix (CCR).          *C
C*****
C
      IF(FIRST)THEN
        CALL COMP(U,W,CCR,M,N,WK11,WK22,WK33,WK44)
      ENDIF

C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C*****
C*      DEFNT computes the definiteness (TYPE) of the matrix which *C
C*      is the difference of the damping matrix (C) and the       *C
C*      critical damping matrix (CCR).                             *C
C*      NN is the dimension of a vector formed by the             *C
C*      aforementioned difference (C - CCR).                       *C
C*      The system description is printed out by DEFNT at this time.*C
C*      i.e. it is stated which type of system (overdamped,       *C
C*      critically damped, underdamped, or mixed) is present.     *C
C*****
C
      CALL DEFNT(C,CCR,TYPE,N,NN,WK11,WKL)

C
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C*****
C*      EIGQUE Asks the user if he wants to compute the damped    *C
C*      eigenvalues. DEVAL is true if answer is yes.             *C
C*      DMPEIG computes them.                                      *C
C*****
C
      CALL EIGQUE(DEVAL)
      IF(DEVAL)THEN
        CALL DMPEIG(M,C,K,LAMDA,N,N2,NN2,WK1,WKL,WK11,WK22,WK11C)
      ENDIF

C
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C*****
C*      OUTFIL creates an output file with the users choice of    *C
C*      values, such as any of the system matrices.              *C
C*****
C
      IF(FIRST)CALL OPNFIL(OPEN,FILEN)
      IF(OPEN)CALL OUTFIL(M,C,K,CCR,U,W,N,TYPE,LAMDA,N2,DEVAL)

C
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
      FIRST=.FALSE.
      ENDIF

C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C*****
C*      RERUN asks the user if he wishes to rerun the problem    *C

```


[illegible]

[illegible]

[illegible]

[illegible]

```

C
C                                     Subroutine CHKC
C
C      CHKC determines if the C matrix is legal, meaning it
C      must be symmetric and either positive definite or positive
C      semi-definite.
C
C      *Varibale List*
C      C - Damping matrix
C      WK - Work matrix
C      LOGICALS - .TRUE. meaning
C      OK - C matrix is legal
C      PD - C matrix is positive definite
C      PSD - Cmatrix is positive semi definite
C
C      SUBROUTINE CHKC(C,N,WK,NN,OK,PSD)
C
C      REAL*8 C(N,N),WK(NN)
C      LOGICAL OK, PSD, PD
C
C      OK=.TRUE.
C      PSD=.FALSE.
C
C      CALL CHKSYN(C,N,OK)
C      IF(OK)THEN
C        CALL PDCHK(C,N,WK,NN,PD)
C        IF(.NOT.PD)THEN
C          CALL PSDCHK(C,N,WK,NN,PSD)
C          IF(.NOT.PSD)THEN
C            OK=.FALSE.
C            PRINT*,' C is not positive semi-definite.'
C            PRINT*,' Illegal system.'
C          ENDIF
C        ENDIF
C      ELSE
C        PRINT*,' C is not symmetric.  Illegal system.'
C      ENDIF
C
C      RETURN
C      END
C
C

```

[illegible]

[illegible]

```

C
C
C
SUBROUTINE SORT(W,E,M)
REAL*8 W(M),E(M,M)
MM=M-1
DO 40 I=1,MM
DO 30 J=1,MM
IF(W(J).LE.W(J+1)) GO TO 25
N=J+1
WTEMP=W(J)
W(J)=W(N)
W(N)=WTEMP
DO 20 K=1,M
ETEMP=E(K,J)
E(K,J)=E(K,N)
E(K,N)=ETEMP
20 CONTINUE
25 CONTINUE
30 CONTINUE
40 CONTINUE
RETURN
END
C
C
C
SUBROUTINE NORM(U,M,N,UT,UTH,UTHU)
REAL*8 U(N,N),M(N,N)
REAL*8 UT(N,N),UTH(N,N),UTHU(N,N)
CALL DMTRN(UT,U,N)
CALL DMMLT(UTH,UT,M,N,N,N)
CALL DMMLT(UTHU,UTH,U,N,N,N)
DO 10 J=1,N
DO 20 I=1,N
U(I,J)=U(I,J)/DSQRT(UTHU(J,J))
20 CONTINUE
10 CONTINUE
C
C
RETURN
END
C
C
C

```

[illegible]

```

CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C                               Subroutine DEFNT                               C
C
C   DEFNT determines the type of damping present in the                      C
C   system.                                                                    C
C   1)C-Ccr=0; critically damped                                              C
C   2)C-Ccr positive definite; overdamped                                    C
C   3)C-Ccr negative definite; underdamped                                    C
C   4)C-Ccr indefinite; mixed damping                                         C
C   The definiteness of the matrix is determined by examining                C
C   the determinant's of the submatrices: sizes 1 X 1 through                C
C   N X N.                                                                     C
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C   *Variable List*                                                            C
C   C - Damping matrix.                                                         C
C   CCR - Critical damping matrix.                                              C
C   DIF - Matrix that is the difference of C - CCR                           C
C   TYPE - Type of definiteness.                                                C
C   N - Size of matrices.                                                       C
C   DET - Determinant of submatrices.                                           C
C   MS - Size of submatrix.                                                      C
C   DPR - Product of diagonal terms of submatrix.                             C
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C   SUBROUTINE DEFNT(C,CCR,TYPE,N,NN,DIF,TH)                                    C
C
C   REAL*8 C(N,N),CCR(N,N),DIF(N,N),TH(NN),DET,DPR                            C
C   CHARACTER*6 TYPE                                                            C
C   LOGICAL QUITP,QUITN                                                         C
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C   PRINT*,' '                                                                    C
C
C   ---Compute C - Ccr.                                                         C
C
C   CALL DMSUB(DIF,C,CCR,N,N)                                                    C
C
C   TYPE='INDEF'                                                                C
C   QUITP=.FALSE.                                                                C
C   QUITN=.FALSE.                                                                C
C
C   ---Check to see if C - Ccr = 0., thus critically damped.                  C
C
C   NC=0
C   DO 10 I=1,N
C   DO 20 J=1,N
C   IF(DABS(DIF(I,J)).LE..0001)NC=NC+1
20  CONTINUE
10  CONTINUE
C   IF(NC.EQ.N**2)THEN
C   TYPE='ZERO'
C   QUITP=.TRUE.
C   QUITN=.TRUE.
C   ENDIF
C

```

[illegible]

[illegible]

1

[illegible]

[illegible]

```

CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C          Subroutine OUTFIL                                     C
C
C      OUTFIL presents the user with the menu of options that   C
C      can be printed to the output file.                       C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C      SUBROUTINE OUTFIL(M,C,K,CCR,U,W,N,TYPE,LAMDA,N2,DEVAL)    C
C
C      REAL*8 M(N,N),C(N,N),K(N,N),CCR(N,N),U(N,N),W(N)         C
C      COMPLEX*16 LAMDA(N2)                                       C
C      CHARACTER*1 RESP(16)                                       C
C      CHARACTER*6 TYPE                                           C
C      INTEGER COL(8)                                             C
C      CHARACTER*32 TEXT                                          C
C      LOGICAL DEVAL,CONT                                         C
C
C      CONT=.TRUE.                                               C
C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C      WRITE(12,'(//)')
C      IF(TYPE.EQ.'ZERO')THEN
C        WRITE(12,10)
C        WRITE(12,*) 'Critically damped system'
C        WRITE(12,10)
C      ELSEIF(TYPE.EQ.'NEGDEF')THEN
C        WRITE(12,20)
C        WRITE(12,*) 'Underdamped system'
C        WRITE(12,20)
C      ELSEIF(TYPE.EQ.'INDEF')THEN
C        WRITE(12,30)
C        WRITE(12,*) 'System with mixed damping'
C        WRITE(12,30)
C      ELSEIF(TYPE.EQ.'POSDEF')THEN
C        WRITE(12,40)
C        WRITE(12,*) 'Overdamped system'
C        WRITE(12,40)
C      ENDIF
10    FORMAT(26('*'))
20    FORMAT(20('*'))
30    FORMAT(27('*'))
40    FORMAT(19('*'))
C
C      PRINT*,'The following information can be printed'
C      PRINT*,'      to your output file.'
C      PRINT*,'      1)Mass Matrix'
C      PRINT*,'      2)Stiffness Matrix'
C      PRINT*,'      3)Damping Matrix'
C      PRINT*,'      4)Critical Damping Matrix'
C      PRINT*,'      5)Undamped Natural Frequencies'
C      PRINT*,'      6)Modal Matrix'
C      IF(DEVAL)PRINT*,'      7)Damped Eigenvalues'
C      PRINT*,' '
C      PRINT*,'Enter the option numbers desired in a single line'
C      PRINT*,' separated by commas. '
C      PRINT*,' '

```



```

      IF(1B.EQ.1)THEN
        WRITE(12,940)(C(I,J),J=1,NCOL)
      ELSE
        WRITE(12,950)(C(I,J),J=5*IB-4,NCOL)
      ENDIF
320    CONTINUE
      WRITE(12,'(' ')')
300    CONTINUE
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C   Write Critical damping matrix                                     C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
      ELSEIF(RESP(1).EQ.'4')THEN
        WRITE(12,'(' ')')
        WRITE(12,*) 'CRITICAL DAMPING MATRIX'
        NI=(N/5)*5
        IF(NI.EQ.N)NBLK=N/5
        IF(NI.NE.N)NBLK=N/5+1
        DO 400 IB=1,NBLK
          IF(N.LE.(5*IB))THEN
            NCOL=N
            NUM=5-(5*IB-N)
          ELSEIF(N.GT.(5*IB))THEN
            NCOL=5*IB
            NUM=5
          ENDIF
          DO 410 J=5*IB-4,NCOL
            JJ=J-5*(IB-1)
            COL(JJ)=J
410          CONTINUE
            IF(1B.EQ.1)THEN
              WRITE(TEXT,900)NUM
              WRITE(12,TEXT)
              WRITE(TEXT,910)NUM
              WRITE(12,TEXT)(COL(JJ),JJ=1,NUM)
            ELSE
              WRITE(TEXT,920)NUM
              WRITE(12,TEXT)
              WRITE(TEXT,930)NUM
              WRITE(12,TEXT)(COL(JJ),JJ=1,NUM)
            ENDIF
            DO 420 I=1,N
              IF(1B.EQ.1)THEN
                WRITE(12,940)(CCR(I,J),J=1,NCOL)
              ELSE
                WRITE(12,950)(CCR(I,J),J=5*IB-4,NCOL)
              ENDIF
420            CONTINUE
              WRITE(12,'(' ')')
400            CONTINUE
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C   Write undamped natural frequencies                             C
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
      ELSEIF(RESP(1).EQ.'5')THEN
        WRITE(12,'(' ')')
        WRITE(12,*) 'UNDAMPED NATURAL FREQUENCIES (rad/sec)'
        WRITE(12,*) ' '
        DO 500 I=1,N
          WRITE(12,510)I,W(I)
510         FORMAT(12,IX,E13.6)

```

C

[illegible]

[illegible]

[illegible]

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