

KINTON B. STEVENS



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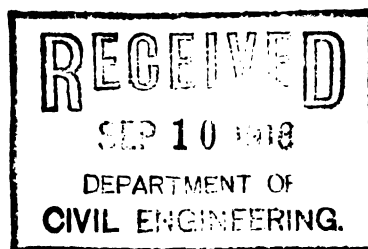
THESIS
BOILER TRIALS

K. B. STEVENS E. H. ADAMS
1906

This thesis was contributed by

Mr. K. B. Stevens

under the date indicated by the department, to replace the original which was destroyed in the fire of March 5, 1916.



THESIS

Boiler Trials

on the

1. horizontal, internal furnace, marine boilers
at the power plant

of the

Michigan Agricultural College.

M. P. Stevens

C. A. Adams

1906.

THESIS

12-4-51
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POWER PLANT

264836

Object of trial.

Under actual running conditions:

- (1) To compare the equivalent evaporation of the boilers.
- (2) To compare their thermal efficiencies.
- (3) To determine excess of air supplied to furnaces by.(a) Pitot tube.
(b) Chemical analysis.

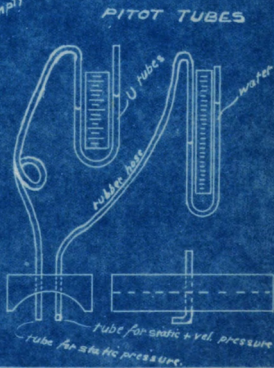
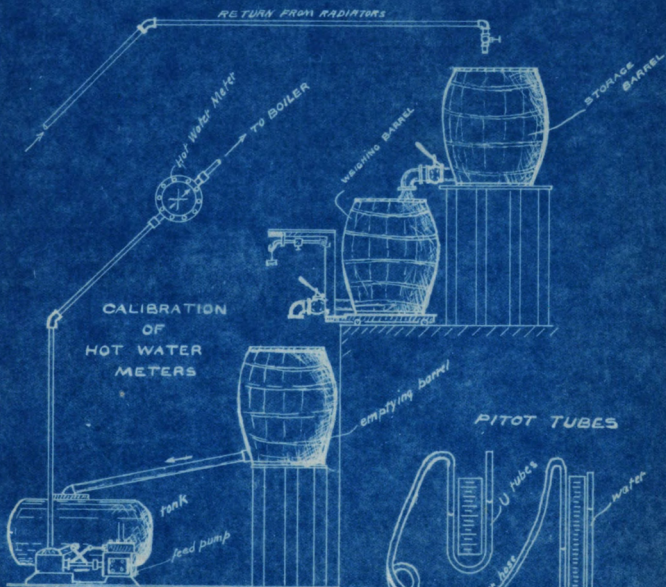
Description of Plant.

These tests were made at the new power house at the Michigan Agricultural College on the boiler plant in Feb. 1908.

The plant is equipped with four horizontal marine boilers manufactured by the Springfield Mfg. Co. The coal for supplying these boiler falls from the bins into a car in an elevator pit. The car, when filled, is lifted by the elevator to a covered train-way, along which the attendant moves it. At the hoppers, the hinged sides of the car are loosened allowing the coal to empty itself. From each hopper the coal falls into an under-feed stroker, which puts it into the furnace.

Forced draft is supplied by an American Blower Co., 11 inch three quarter house bottom horizontal discharge fan which is driven direct connected by a type "A", A.C.M., piston valve engine.

This engine is driven with steam from the main pipe. The pipe leading steam to the engine is equipped with a valve, being thus throttled lessens its speed, and less air is supplied to the furnaces. Driven by belt from the fan engine is a Cole auto, which regulates the rate at which the Jones under-feed



stokers work. As the speed of the fans varies, the rate of the stoking is changed. This means that when the air supply is cut down by the throttling of the fan engine, that the rate of coal supply is cut down in the same ratio. Thus the system is perfectly self regulating; and the steam pressure remains practically constant, as may be seen from the data sheets.

Description of tests.

It was necessary, in order to make a comparative test on the boilers, to test one of the battery in detail each day, and the others together as a unit. This made it impossible for us to determine what each individual boiler was doing, and the results obtained from the three coupled together each day would not be affected in case the weather or other conditions varied.

Apparatus.

Our testing apparatus consisted of two hot water meters, (a 11" Hersey and a 5" National meter) 2 compensators separating calorimeter, barrels for the weighing of water in the meters & dilution, a thermometer, thermocouple, speed counter, Pitot tubes, scales, rheobancors, etc.

Water.

Our meters were calibrated by weighing the water in barrels equipped with quick opening valves. Both were calibrated under actual test conditions. The return water from the calorimeters, which was used as feed water, ran into a float tank from which it was pumped, part through the small meter to the single boiler, and the remainder through the Hersey meter to the gas -

al feed of the three boilers running together. The temperature of the feed water varied considerably, because it was necessary from time to time to supply cold water to the float tank in order to replace the water lost by leakage etc.

The coal.

We could not use the coal hoppers, because it would be very difficult to weigh the coal placed in them from above, so we avoided them by sliding the shoots up to the foot of the hoppers, thus preventing any chance for loose coal falling into the stokers. From the extra we drew and weighed our coal in large steel barrels, depositing it in piles near each boiler, from which it was shoveled into the stokers. The amount of coal used for the test of the single boiler was of course kept separate from the other three. At the close of each days test whatever coal remained on the floor was weighed back and deducted.

The air.

The air is supplied to the boilers through two nine inch ~~tubes~~ galvanized iron pipes, one entering on each side below the fire grate. As the change in steam pressure caused a change in the speed of the fan, we had considerable variations of the pressure in the air pipes. The readings on the manometer were taken every half hour, and the revolutions of the blower at the same time. Our apparatus for determining the pressure of the air and thence the velocity, consisted of two U shaped tubes graduated to tenths of inches, connected each by a large rubber tube to our Pitot tubes (see fig.) the straight tube gave the

static pressure in the pipe, and the ^{right} ~~left~~ angled tube gave the static; plus the pressure due to the velocity of the air.

Thus the difference of reading between the two would be the pressure caused by the velocity of the air, so by the formula

$V = \sqrt{2P}$ we obtain the velocity of the air due to the boilers.

Knowing the size of the air pipe, we may then obtain the volume of air supplied to the boilers. It is then reduced to the pounds of air per pound of coal. This was also determined by the analysis of the flue ~~gas~~ gas made by the chemical Dept.

static pressure in the pipe, and the ^{right} ~~left~~ angled tube gave the static plus the pressure due to the velocity of the air.

Thus the difference of reading between the two would be due to pressure caused by the velocity of the air, so by the formula $V = \sqrt{\frac{2P}{\rho}}$ we obtain the velocity of the air due to the boilers. Knowing the size of the air pipe, we may then obtain the volume of air supplied to the boilers. It was then referred to the pounds of air per pound of coal. This was also determined by the analysis of the flue gas as made by the chemical Dept.

BOILERS No 1 & No 2 R-3-4. LOG OF READINGS

FEB 24 - 1906

TIME	WT. of COAL		WT. of Water		TEMPERATURE			PRESSURES			R.P.M. of Governor	Barometer
	For Single Boiler	For Three Boilers	For Single Boiler	For Three Boilers	Feed Water	Room	Outside	Static in Air Pipe	Velocity Air Pipe	Gage		
8. ^{AM}					151							
8. ³⁰					152	82	40	.8	1.7	80	244	29.2
9. ⁰⁰					173	95	40	.8	1.8	"	230	29.2
9. ³⁰					177	85	43	1.2	2.	"	255	29.1
10. ⁰⁰					180	86	46	1.	2.	"	270	29.18
10. ³⁰					176	85	47	1.1	2.	"	270	29.18
11. ⁰⁰					140	85	48	1.	1.9	"	255	29.16
11. ³⁰					128	85	48	1.3	2.1	"	283	29.16
12. ⁰⁰					190	86	48	.8	1.1	"	203	29.12
12. ³⁰					180	82	50	.9	1.2	"	215	29.1
1. ⁰⁰					122	82	50	1.1	1.4	"	226	29.1
1. ³⁰					105	82	48	1.1	1.5	"	232	29.1
2. ⁰⁰					135	83	47	1.2	2.	"	265	29.08
2. ³⁰					150	81	47	1.1	1.5	"	234	29.06
3. ⁰⁰					153	86	47	1.	1.4	"	234	29.06
3. ³⁰					152	85	47					
AVC.	2856 [#]	8573 [#]	23440 [#]	67576 [#]	154°	84	47	1.03"	1.54"	80 [#]	245	29.12

Unmix. water evap. per lb. coal

from air at 40°

0.11

0.25 #

Water evap. per lb. combustibles

0.15

0.28

Unmix. H₂O per lb. combustible from

expt at 40°

15.58 #

1.1 #

H₂O in air at 40°

20.6

227.4

air per lb. coal by pitot tube

10.96 #

air reqd. for complete combustibles

10.84 #

10.84 #

air per lb. coal from gas analysis

8.91 #

LOG OF READINGS

FEB. 21, 1906

TIME.	WT. of Coal		WT. of Water		Temperatures			Pressures			R.P.M.	Barometer
	For Single # Boiler No 2	For Three # Boilers Nos. 1, 3, 4	No. 2. Single Boiler	No. 1, 3, 4 For Three Boilers	Feed Water °F	Room	outside	Static in air pipe	Velocity in air pipe	Gage		
7.30												
8.00					124	74	35	1.9	2.1	80	240	29.14
8.30					123	81	37	1.8	2.0	"	242	29.13
9.00					126	81	37	1.8	1.9	"	250	29.13
9.30					90	83	39	2.6	2.8	"	300	29.16
10.00					200	85	40	1.2	2.0	"	245	29.16
10.30					204	84	40	1.2	1.6	"	224	29.18
11.00					138	82	41	1.0	1.6	"	264	29.18
11.30					150	83	40	.8	1.2	"	200	29.18
12.00					150	83	42	.8	1.3	"	208	29.2
12.30					153	85	42	1.0	1.5	"	224	29.19
1.00					147	85	40	1.0	1.8	"	254	29.2
1.30					149	86	40	1.3	1.9	"	259	29.2
2.00					150	86	39	1.4	2.00	"	265	29.22
2.30					140	85	39	.9	1.7	"	240	29.23
3.00					172	83	37	1.0	1.5	"	243	29.25
3.30					160	79	37	1.0	1.7	"	242	29.25
4.00					135	84	37	1.2	1.8	"	240	29.26
4.30					146	84	36	1.0	1.3	"	203	29.28
5.00					153	82	35	.8	1.1	"	200	29.29
5.30					151	80	35	.6	1.0	"	170	29.30
6.00					130	80	34	.6	1.1	"	180	29.30
6.30					134	85	34	.8	1.1	"	190	29.32
7.00					165	85	34	.8	.9	"	192	29.32
7.30										"		
Arg.	4281*	14564*	31800*	117283*	146	82	37	1.06	1.6	80	230	29.23

ONLY INITIAL AND FINAL READINGS TAKEN

Boiler No. 1.

and

Boilers Nos. 1,2,4

May 1902

	Boiler No. 1.	Boilers Nos. 1,2,4
Kind of coal	Vainmont	Shael.
" " trial	Alternate	Method
Duration of trial	16 hrs.	16 hrs.
ave. steam press. in boilers	85 °	85 °
Temp. of external air	57 °	57 °
" " room	68 °	70 °
" " feed water	145 °	145 °
dry coal consumed	14534 #	14534 #
Moisture in coal (%)	1.05	1.05
dry moisture consumed	1536 #	1541 #
total dry refuse	337.5 #	315 #
total combustible	14196 #	14219 #
dry coal consumed per hr.	908 #	901 #
combustible " " "	912 #	900 #
quality of steam	98.6%	98.6%
No. heat units per lb. coal	13600	13600
heat units absorbed per lb.		
steam from and at 212 °	1395	1460
Efficiency of boiler	86.4	87.1
Total water evaporated	11000 #	11000 #
equiv. water from and at 212 °	11000 #	11000 #
" " " per hr. from and at 212 °	687 #	687 #

Boiler No. 1

and

Boilers Nos. 1, 5, 6

Boilers 1, 5, 6

water evap. per lb. of coal	9.24%	895%
Equiv. water evap. from coal at 212 F	8.19%	8.91%
water evap. per lb. combustible	8.81%	8.1%
Equiv. water evap. per lb. from coal at 212 F	8.19%	8.91%
H.P.A.S.I.V.	97.4	245.5
air per lb. of coal by pitot tube	21.88%	
air req. for complete combustion	10.74%	
air per lb. coal from gas analysis	27.1%	

LOG OF RESULTS

FEB. 22-1906

Time	Wt. of Coal		Wt. of Water		Temperatures			Pressures			R.P.M.	Barometer.
	For Single Boiler No 3	For Three Boilers Nos 1, 2, 4	No 3 For Single Boiler	No 1, 2, 4 For Three Boilers	Feed Water	Room	outside	Static in air pipe	Velocity in air pipe	Gage		
7.30												
8.00					170	68	30	.9	1.6	80	239	29.5
8.30					168	76	31	.7	1.1	"	200	29.5
9.00					147	78	31	1.0	1.5	"	242	29.5
9.30					166	78	32	1.0	1.4	"	239	29.5
10.00					150	80	32	1.0	1.5	"	225	29.5
10.30					153	78	33	1.1	1.4	"	238	29.5
11.00					150	78	35	1.0	1.3	"	220	29.5
11.30					155	80	35	1.1	1.5	"	231	29.5
12.00					160	80	35	1.2	1.7	"	250	29.47
12.30					170	84	36	1.0	1.7	"	246	29.49
1.00					166	83	36	1.0	1.5	"	238	29.41
1.30					164	86	36	1.0	1.4	"	222	29.40
2.00					163	86	36	1.0	1.5	"	230	29.40
2.30					153	85	39	.6	.8	"	165	29.4
3.00					150	85	40	1.0	1.3	"	205	29.39
3.30					158	84	41	1.0	1.5	"	229	"
4.00					177	82	41	1.0	1.5	"	222	29.37
4.30					171	82	41	.9	1.2	"	204	"
5.00					155	82	39	.8	1.1	"	190	"
5.30					172	80	37	1.0	1.2	"	209	29.36
6.00					162	81	37	.9	1.2	"	210	"
6.30					154	80	37	.9	1.1	"	202	29.35
7.00					185	80	37	.6	.8	"	185	"
7.30												
Avg.	4771 [#]	13433 [#]	39880 [#]	10790 [#]	162	80	36	.9	1.33	80	219	29.42

ONLY INITIAL AND FINAL READINGS TAKEN

Boiler No. 3

and

Boilers No. 1, 2, 4

	Boiler No. 3 Fairmont	Boilers No. 1, 2, 4 Clack
Kind of fuel	bitum.	slack
" " test	17 hrs.	12 hrs.
Duration of trial	86.4	82.7
Av. steam pres. by gauge	76 F	76 F
Temp. of external air	89 F	89 F
" " room	108 F	108 F
" " feed water	177.1 F	174.3 F
moist coal consumed	165 #	1.35 #
moisture in coal (%)	4771 #	13302 #
dry coal consumed	451 #	1201 #
total dry refuse	1222 #	12225 #
total combustible	797.6 #	1119 #
Dry coal consumed per hr.	302.3 #	1606 #
Combustible consumed per hr.	89.1 #	109.1 #
quality of steam	12020	12020
No. heat units in one lb. coal	1136	1102
Heat units per lb. steam from end at 21 F	87.5	84.0
Eff. of boiler	79330 #	167007 #
Total H ₂ O evap.	4741	117100
equiv. H ₂ O from end at 212	5321	12627
" " evap. per hr.	3.77	3.07

over

LOG OF READINGS

Boilers No 4 & Nos 1-2-3.

FEB. 23, 1906

TIME	WT. of COAL		WT. of Water		Temperatures			Pressures			R.P.M. of blower	Barometer
	For Single Boiler.	For Three Boilers	For Single Boiler	For Three Boilers	Feed Water	Room	Outside	Static in air pipe	Velocity in air pipe	Gage		
7:30												
8:00					173	87	39	.8	1.3	80	223	2922
8:30					136	89	40	1.1	2.2	"	273	2922
9:00					132	89	45	1.1	2.	"	268	2922
9:30					160	90	48	1.1	1.9	"	262	2922
10:00					150	93	51	1.	1.6	"	248	2923
10:30					136	93	51	1	1.7	"	252	2924
11:00					129	93	55	.8	1.5	"	239	2924
11:30					121	92	56	1.	1.8	"	262	2923
12:00					198	92	56	.8	1.3	"	240	2923
12:30					197	95	56	.8	1.6	"	248	2924
1:00					158	97	56	1.1	2.	"	282	2923
1:30					144	99	60	1.2	2.2	"		2922
2:00					99	96	60	1.4	3.1	"	350	2924
2:30					148	96	60	1.2	2.	"	265	2924
3:00					131	94	60	1.	1.5	"	230	2923
3:30					160	95	58	.5	.9	"	181	2926
4:00					145	95	55	4	.7	"	166	2930
4:30					130	94	52	.9	1.1	"	210	293
5:00					189	93	48	.6	.8	"	185	293
5:30					158	93	48	.5	.8	"	183	2932
6:00					190	93	48	.6	.9	"	180	2932
6:30					161	92	46	.6	.9	"	182	2932
7:00										"		
7:25												
Ave.	4642 ²	13144 ²	37080 ²	108644 ²	159	93	54	.945	1.64	80	237	2927

ONLY INITIAL AND FINAL READINGS TAKEN

Boiler No. 4
and
Boilers Nos. 1, 2, 3

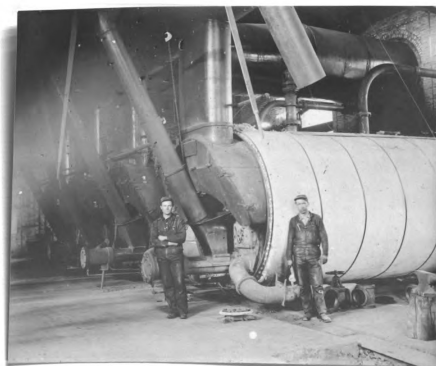
	Boiler No. 4	Boilers Nos. 1, 2, 3
Kind of fuel	Wainmont	Clack
" " trial	Alternate	Let'ed
Duration of trial	12 hrs.	12 hrs.
steam pres. of boilers	80 $\frac{1}{2}$	80 $\frac{1}{2}$
temp. of external air	59 $^{\circ}$ F	56 $^{\circ}$ F
" " room	93 $^{\circ}$ F	93 $^{\circ}$ F
" " Feed water	159 $^{\circ}$ F	159 $^{\circ}$ F
moist coal consumed	13149 $\frac{1}{2}$	13144 $\frac{1}{2}$
moisture in coal (%)	1.65	1.65
dry coal consumed	4594 $\frac{1}{2}$	4596 $\frac{1}{2}$
total dry refuse	419 $\frac{1}{2}$	1189 $\frac{1}{2}$
total combustible	4130 $\frac{1}{2}$	11300 $\frac{1}{2}$
dry coal consumed per hr.	373 $\frac{1}{2}$	1949 $\frac{1}{2}$
combustible consumed per hr.	336 $\frac{1}{2}$	154 $\frac{1}{2}$
quality of steam	97.7%	93.7%
No. of heat units in 1 lb. of coal	13020	13020
heat units absorbed by boiler per lb. of steam.	1495	1419
efficiency of boiler	64.8%	66.7%
total evap.	27685 $\frac{1}{2}$	16874 $\frac{1}{2}$
equiv. water from cond. at 115 $^{\circ}$ F	40391 $\frac{1}{2}$	118622 $\frac{1}{2}$
" " " " " " per hr.	3349 $\frac{1}{2}$	10718 $\frac{1}{2}$

Boiler No. 4

and

Boilers Nos. 1, 2, 3

	Boiler No. 4	Boilers Nos. 1, 2, 3
Water evap. per lb. coal	7.93 $\frac{1}{2}$	8.13 $\frac{1}{2}$
" " " " coal from and at 212 F	8.76 $\frac{1}{2}$	8.99 $\frac{1}{2}$
" " " " combustible	8.79 $\frac{1}{2}$	9.18 $\frac{1}{2}$
Water evap. per lb. combustible		
from and at 212 F	9.66 $\frac{1}{2}$	9.98 $\frac{1}{2}$
H.P. A.S. L.T. rating	97.4	275.9
Air per lb. coal by pilot tube	8.1 $\frac{1}{2}$	
Air per lb. coal by gas analysis	79.9 $\frac{1}{2}$	80.9 $\frac{1}{2}$



RESULTS ARRANGED FOR COMPARISON

Number of Boiler	Heat units per lb of steam from and at 212° F.	Efficiency of Boilers	Water evaporated per lb. of coal	Equivalent evaporation per lb of combustible	Avg. Horse Power developed.	Air per lb. of Coal	
						By Pitot tube	By Gas Analysis
1	1388	67 %	8.23	10.02	99.6	28.9 [#]	39.4 [#]
2-3-4	1508	63.1%	7.87	9.64	95.6	—	—
2	1595	60.6%	7.42	9.12	93.4	31.88 [#]	47 [#]
1-3-4	1466	65.9%	8.05	9.89	115	—	—
3	1430	67.5%	8.37	10.11	104.8	25.6 [#]	30.6 [#]
1-2-4	1492	64.9%	8.03	9.7	94	—	—
4	1495	64.5 [%]	7.99	9.66	97.4	33.4	39.9
1-2-3	1449	66.7 [%]	8.26	9.98	91.3	—	—

1000

Year	1900	1901	1902	1903	1904	1905	1906	1907	1908	1909	1910	1911	1912	1913	1914	1915	1916	1917	1918	1919	1920	1921	1922	1923	1924	1925	1926	1927	1928	1929	1930	1931	1932	1933	1934	1935	1936	1937	1938	1939	1940	1941	1942	1943	1944	1945	1946	1947	1948	1949	1950	1951	1952	1953	1954	1955	1956	1957	1958	1959	1960	1961	1962	1963	1964	1965	1966	1967	1968	1969	1970	1971	1972	1973	1974	1975	1976	1977	1978	1979	1980	1981	1982	1983	1984	1985	1986	1987	1988	1989	1990	1991	1992	1993	1994	1995	1996	1997	1998	1999	2000	2001	2002	2003	2004	2005	2006	2007	2008	2009	2010	2011	2012	2013	2014	2015	2016	2017	2018	2019	2020	2021	2022	2023	2024	2025	2026	2027	2028	2029	2030	2031	2032	2033	2034	2035	2036	2037	2038	2039	2040	2041	2042	2043	2044	2045	2046	2047	2048	2049	2050	2051	2052	2053	2054	2055	2056	2057	2058	2059	2060	2061	2062	2063	2064	2065	2066	2067	2068	2069	2070	2071	2072	2073	2074	2075	2076	2077	2078	2079	2080	2081	2082	2083	2084	2085	2086	2087	2088	2089	2090	2091	2092	2093	2094	2095	2096	2097	2098	2099
1900	1901	1902	1903	1904	1905	1906	1907	1908	1909	1910	1911	1912	1913	1914	1915	1916	1917	1918	1919	1920	1921	1922	1923	1924	1925	1926	1927	1928	1929	1930	1931	1932	1933	1934	1935	1936	1937	1938	1939	1940	1941	1942	1943	1944	1945	1946	1947	1948	1949	1950	1951	1952	1953	1954	1955	1956	1957	1958	1959	1960	1961	1962	1963	1964	1965	1966	1967	1968	1969	1970	1971	1972	1973	1974	1975	1976	1977	1978	1979	1980	1981	1982	1983	1984	1985	1986	1987	1988	1989	1990	1991	1992	1993	1994	1995	1996	1997	1998	1999	2000	2001	2002	2003	2004	2005	2006	2007	2008	2009	2010	2011	2012	2013	2014	2015	2016	2017	2018	2019	2020	2021	2022	2023	2024	2025	2026	2027	2028	2029	2030	2031	2032	2033	2034	2035	2036	2037	2038	2039	2040	2041	2042	2043	2044	2045	2046	2047	2048	2049	2050	2051	2052	2053	2054	2055	2056	2057	2058	2059	2060	2061	2062	2063	2064	2065	2066	2067	2068	2069	2070	2071	2072	2073	2074	2075	2076	2077	2078	2079	2080	2081	2082	2083	2084	2085	2086	2087	2088	2089	2090	2091	2092	2093	2094	2095	2096	2097	2098	2099	

Feb. 21, 1953

Year	1900	1901	1902	1903	Location
I. I.	7.0	11.0	0.1	11.0	Littleport F.I.
II. II	1.0	11.0	0	11.0	Littleport F.I.
III. III	1.0	11.0	0	11.0	Littleport F.I.
IV. IV	1.0	11.0	0	11.0	Littleport F.I.
V. V	1.0	11.0	0	11.0	Littleport F.I.
VI. VI	1.0	11.0	0	11.0	Littleport F.I.
VII. VII	1.0	11.0	0	11.0	Littleport F.I.
VIII. VIII	1.0	11.0	0	11.0	Littleport F.I.
IX. IX	1.0	11.0	0	11.0	Littleport F.I.
X. X	1.0	11.0	0	11.0	Littleport F.I.
XI. XI	1.0	11.0	0	11.0	Littleport F.I.
XII. XII	1.0	11.0	0	11.0	Littleport F.I.
Total	12.0	12.0	0	12.0	Littleport F.I.

Table No. 2

May 1, 1934.

Time	Temp	Wind	Bar	Hum	Analyst
12:00	55.0	10.0	30.0	75.0	W. H. H.
12:30	55.0	10.0	30.0	75.0	W. H. H.
13:00	55.0	10.0	30.0	75.0	"
13:30	55.0	10.0	30.0	75.0	"
14:00	55.0	10.0	30.0	75.0	"
14:30	55.0	10.0	30.0	75.0	"
15:00	55.0	10.0	30.0	75.0	"
15:30	55.0	10.0	30.0	75.0	"
16:00	55.0	10.0	30.0	75.0	"
16:30	55.0	10.0	30.0	75.0	"
17:00	55.0	10.0	30.0	75.0	"
17:30	55.0	10.0	30.0	75.0	"
18:00	55.0	10.0	30.0	75.0	"
18:30	55.0	10.0	30.0	75.0	"
19:00	55.0	10.0	30.0	75.0	"
19:30	55.0	10.0	30.0	75.0	"
20:00	55.0	10.0	30.0	75.0	"
20:30	55.0	10.0	30.0	75.0	"
21:00	55.0	10.0	30.0	75.0	"
21:30	55.0	10.0	30.0	75.0	"
22:00	55.0	10.0	30.0	75.0	"
22:30	55.0	10.0	30.0	75.0	"
23:00	55.0	10.0	30.0	75.0	"
23:30	55.0	10.0	30.0	75.0	"
24:00	55.0	10.0	30.0	75.0	"

Table No. 3

May 1, 1934.

Time	Temp	Wind	Bar	Hum	Analyst
12:00	55.0	10.0	30.0	75.0	W. H. H.
12:30	55.0	10.0	30.0	75.0	"
13:00	55.0	10.0	30.0	75.0	"
13:30	55.0	10.0	30.0	75.0	"
14:00	55.0	10.0	30.0	75.0	"
14:30	55.0	10.0	30.0	75.0	"
15:00	55.0	10.0	30.0	75.0	"
15:30	55.0	10.0	30.0	75.0	"
16:00	55.0	10.0	30.0	75.0	"
16:30	55.0	10.0	30.0	75.0	"
17:00	55.0	10.0	30.0	75.0	"
17:30	55.0	10.0	30.0	75.0	"
18:00	55.0	10.0	30.0	75.0	"
18:30	55.0	10.0	30.0	75.0	"
19:00	55.0	10.0	30.0	75.0	"
19:30	55.0	10.0	30.0	75.0	"
20:00	55.0	10.0	30.0	75.0	"
20:30	55.0	10.0	30.0	75.0	"
21:00	55.0	10.0	30.0	75.0	"
21:30	55.0	10.0	30.0	75.0	"
22:00	55.0	10.0	30.0	75.0	"
22:30	55.0	10.0	30.0	75.0	"
23:00	55.0	10.0	30.0	75.0	"
23:30	55.0	10.0	30.0	75.0	"
24:00	55.0	10.0	30.0	75.0	"

Proximate analysis of coal.
 Moisture ----- 11.70%
 Ash ----- 11.11%
 Volatile ----- 1.11%
 Moisture ----- 1.95%
 Sulphur ----- 0.07%
 Fixed carbon ----- 17.06%



Conclusion.

It is noticeable from the results, that boiler 1 and 2 show higher rates of evaporation, and higher efficiencies than boilers 3 and 4. This may be due to the regulation of the dampers. The result also that boilers 1 and 2 were receiving less air; and at the same time were carrying more of a load than the other two. This means that boilers 1 and 2 were losing less heat in the chimney; and were consequently more efficient than the other two.

It is probable that the air results obtained from the chemical analysis of the flue gases are more nearly correct than those obtained by the use of the pitot tubes; since in the latter method we have no way of telling just what per cent is lost in the friction of the tubes. We will therefore consider only the results of the flue gas analysis in the following calculation of the loss from the excess of air. The average amount of air supplied to the boiler per lb. coal was 39.2 lbs.

Of this amount 10.34 % was actually needed for combustion. The excess of air was therefore 279%. Since the temperature of the gases was not taken we will assume a difference of temperature between the chimney and the external air of 200 °.

O ₂	-----	6.01X 22=	132
N	-----	14.2 X 14=	227
C	-----	neglected (very small)	
H	-----	<u>72.2 X 14</u>	<u>1118</u>
Total-----		100 %	1477

$$\frac{132}{1477} \times 20.4 = 1.8 \% \text{ -- wt. of } O_2$$

$$\frac{2.7}{1477} \times 70.2 = 0.0013 \text{ --- wt. of C.}$$

$$\frac{111.2}{1477} \times 70.2 = 5.27 \text{ --- lb. of H.}$$

Weight :	Specific	Temp. Increase	B.T.U.
3.0--3.5 :	.2155	500	: 227.5
3.5--4.0 :	.2175	500	: 292.0
4.0--4.5 :	.2153	500	: 318.0
Total -----			837.5

Heating power of 100 pounds steam 17700 B.T.U.

$$\frac{837.5}{17700} = 0.047$$

There still a difference of only 300° between the hot air and the chimney so still that 21.7% of the heat goes up the chimney.

21.7 --- average efficiency of boiler.

21.7 --- chimney loss.

$$\frac{17.15}{177.5} \text{ --- conduction and radiation (9.6 efficiency)}$$

177.5 --- Total.

From these results, it appears that greater attention should be paid to the regulation of the dampers in the air pipes. The external appearance of the chimney also indicated a heavy mass of the air, very little smoke appearing at the top.

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